

# Design of a Quadrotor Roll Controller Using System Identification to Improve Empirical Results

Anil Ufuk Batmaz, Ovunc Elbir, and Cosku Kasnakoglu

**Abstract**—Unmanned Aerial Vehicles (UAVs), specifically quadrotors, is one of the major topics of study in current literature with applications to numerous fields. In this paper we consider a test-bed for the design of a low cost flight controller for a quadrotor and as a first step in design we demonstrate the design of the roll controller on an experimental setup through the stages of data collection, modeling, control design and verification. The procedure consists of four stages: 1) Experimental determination of controller coefficients, 2) Data collection, 3) System identification, 4) Controller redesign by tuning coefficients with a numerical search. It is observed that the system designed as such is capable of achieving the desired roll stabilization, and coefficient tuning on the identified model noticeably improves the settling time and steady state oscillation amplitude.

**Index Terms**—Aerial vehicle control, discrete time, linear estimation, quadrotor.

## I. INTRODUCTION

Unmanned Aerial Vehicles (UAVs) have recently gained great importance in both military and civil applications [1]-[5]. These designs also call for the development of various control methodologies [6]-[8]. An important subset of UAVs is quadrotors, which have become popular recently due to their small size and maneuverability. Research efforts on quadrotors include attitude stabilization, estimation and multi-vehicle configurations [9]-[11].

In the present work we attempt to demonstrate roll-axis stabilizing controller design approach on an experimental test-bed in an attempt to form a foundation for our ultimate research goals of building a unique quadrotor system from bottom up. For this purpose we built a custom quadrotor hardware and implemented software procedures to drive the servomotors, carry out measurements, communicate data, and control the attitude of the UAV, using reasonable cost and commonly available electronic components. An important contribution of this paper is to illustrate a simple process to improve empirically determined controllers so as to improve the overall closed-loop response of the system. This involves identifying a transfer function around the desired equilibrium (which is the horizontal axis for roll stabilization) and performing a numerical search using this model to tune the coefficients. The rest of the paper will explain the methodology, present experimental results and discuss the findings.

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## II. METHODOLOGY

The basic quadrotor model used in the study is shown in Fig. 1.  $F_1, F_2, F_3$  and  $F_4$  are the forces applied by the servomotors, as result of which the pitch ( $\theta$ ), roll ( $\phi$ ) and yaw angle ( $\psi$ ) of the quadrotor are produced.

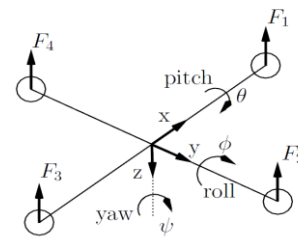


Fig. 1. Simple Quadrotor Model

## III. HARDWARE DESIGN

The general overview of the hardware design can be seen in Fig. 2.

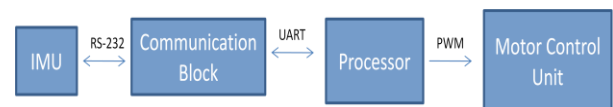


Fig. 2. General overview of the hardware design.

For the IMU we utilize Microstrain-3DM-GX2, which contains a triaxial accelerometer, triaxial gyro, triaxial magnetometer, and an on-board processor running a sensor fusion algorithm. For the communication block we implement a voltage level converter in order to map the output of the IMU into UART voltage levels. As for the microprocessor, a PIC32MX795F512L has been used which can operate up to 80 MHz. The test platform hardware is shown in Fig. 3. As mentioned previously, for the sake of example we shall only consider the stabilization of the roll angle  $\phi$ ; hence, the quadrotor has been fixed in the setup to allow for rolling behavior only.

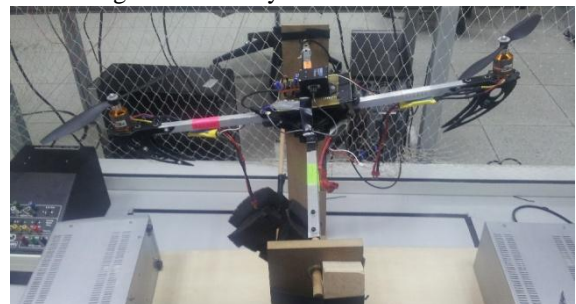


Fig. 3. Experimental quadrotor test platform.

#### IV. SOFTWARE DESIGN

The roll angle  $\phi$  is calculated from the outputs of the IMU using the following equation

$$\phi = \frac{a_y}{a_x \sin \theta - a_z \cos \theta} \quad (1)$$

where  $a_x$ ,  $a_y$  and  $a_z$  are the accelerations read from the inertial sensors in the  $x$ ,  $y$ ,  $z$  axis respectively and  $\theta = \tan^{-1}\left(\frac{a_x}{a_z}\right)$ . This calculated angle is then processed by a Kalman filter so as to obtain a cleaner and more precise measurement. The process and measurement noise covariances of the filter were obtained empirically as  $Q = 1$  and  $R = 1.5$ . Following filtering, the measurements enter a PID controller, whose block diagram is illustrated in Fig. 4.

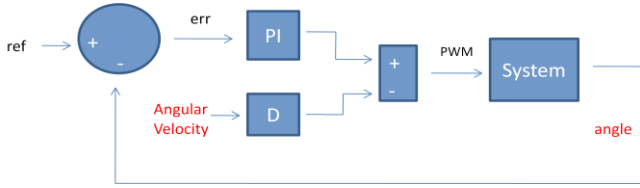


Fig. 4. Block diagram of the PID Controller

Note from the figure that for better numerical accuracy, the derivative term is used directly from the angular velocity measurements obtained from the IMU instead of numerically differentiating the error signal. Note also the minus sign in front of the D term since for a constant roll angle reference we have  $\frac{derr}{dt} = \frac{d}{dt}(\text{ref} - \text{err}) = -\frac{d}{dt}\text{err}$ . The method we employ in determining the PID coefficients consists of four stages: 1) Experimental determination of the PID coefficients, 2) Data collection, 3) System identification, 4) Controller redesign by tuning coefficients with a numerical search. For the first stage we run the experimental setup and use heuristic rules of thumb, such as slowly increasing P until the system somewhat oscillates around the horizontal, then adding a D term to reduce oscillations and finally adding an I term to eliminate the steady state error. The values resulting from this procedure are  $K_p = 0.5$ ,  $K_i = 0.001$  and  $K_d = 0.45$ , which produced in a closed loop system capable of stabilizing the roll angle, but the response was slower than desired and quite jittery. For this reason we proceed with the extra tasks described below.

#### V. SYSTEM IDENTIFICATION

We perform system identification using experimental data obtained from the PID coefficients mentioned above with the goal of producing a linear model around the operating point  $\phi \approx 0$ . The system input is selected as the mean-shifted pulse width modulation value (PWM) that is fed to the servomotors rotating the propellers. The output is the roll angle  $\phi$ . Numerous system identification techniques were applied to the data through the use of MATLAB System Identification Toolbox, but the best results were achieved with subspace identification (N4SID) [12]. The discretized version of this model has the transfer function

$$G(z) = \frac{0.02526z^{-1} - 0.02892z^{-2} - 0.01516z^{-3} + 0.01881z^{-4}}{1 - 3.415z^{-1} + 4.299z^{-2} - 2.352z^{-3} + 0.4682z^{-4}} \quad (4)$$

where the sampling period is  $T_s = 0.025$  s, which is the rate that we process data for our particular hardware/software configuration. One can observe a good agreement between the output measured from experiments and the output simulated from the model, which are compared in Fig. 5.

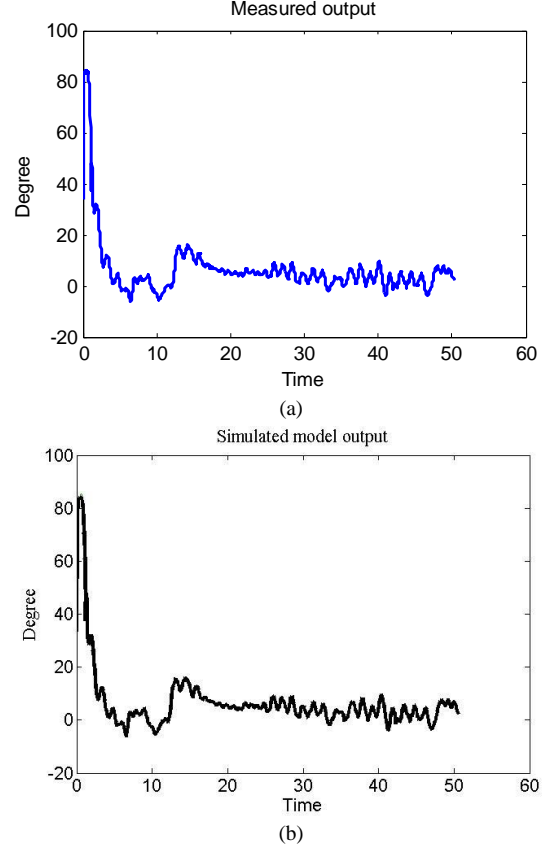


Fig. 5. Measured output (a) and simulated model output (b).

#### VI. CONTROLLER REDESIGN

With the model of the system at hand, we now proceed to redesign the controller so as to improve the performance of the closed loop system, in particular, the time it takes for the system to settle. For this purpose we set up a numerical search problem in MATLAB within a five percent neighborhood of the empirical coefficient values, and look for a solution that minimizes the settling time. This process yields the coefficient values  $K_p = 0.5259$ ,  $K_i = 0.081$  and  $K_d = 0.53$ , using which we form the discrete time PID controller

$$C(z) = K_p + K_i \frac{1+z^{-1}}{1-z^{-1}} \frac{T_s}{2} + K_d \frac{1-z^{-1}}{T_s} \quad (3)$$

where the derivation and integration in discrete time are approximated as  $y(k) = \frac{u(k)-u(k-1)}{T_s}$  and  $y(k) = y(k-1) + \frac{u(k)+u(k-1)}{2} T_s$  respectively.

#### VII. RESULTS

Fig. 6 shows a comparison between two sets of the experimental results: the response of the closed-loop system with the empirical PID coefficients and the response of the closed-loop system with the PID coefficients tuned on the model obtained from system identification. It can clearly be seen that the tuned coefficients have improved the

closed-loop response significantly; the roll angle settles much faster, and once amplitude of the steady state oscillations are lower.

### VIII. CONCLUSION AND FUTURE WORK

This paper presented a quadrotor roll axis control system based on a PID controller. The controller was first tuned experimentally so that the response stays around the horizontal, after which mean-shifted servomotor PWM values and the roll angle readings from the IMU were stored to form an input-output data set. This set was subjected to system identification so as to produce a transfer function of the quadrotor system around the origin, and it was observed that the model can reproduce the system response quite acceptable. Using a numerical search procedure, the PID coefficients were then tuned around a local neighborhood of the empirical PID for a faster settling time. The coefficients obtained from this procedure were tested on the experimental setup and it was observed that the settling time as well as the steady state oscillations of the closed-loop system was improved.

Future research directions include extending the results to the control of the pitch and yaw axis, as well as the testing of the approach presented on different air vehicles.

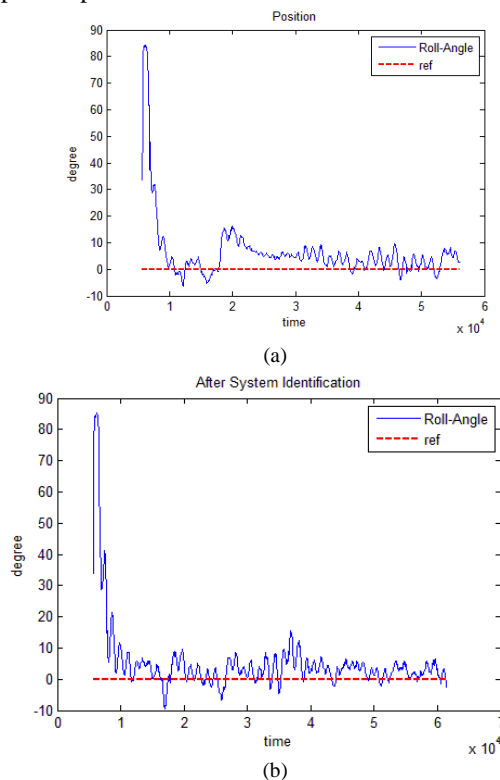


Fig. 6. Experimental results showing the roll angle  $\phi$  vs. reference  $0^\circ$  with empirical PID coefficients (a) and PID coefficients tuned on the identified model (b).

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